Effects of Hardness and Crack Geometries on ΔK_{th} of Small Cracks Emanating from Small Defects

REFERENCE Murakami, Y. and Endo, M. Effects of Hardness and Crack Geometries on ΔK_{th} of Small Cracks Emanating from Small Defects, The Behaviour of Short Fatigue Cracks, EGF Pub. 1 (Edited by K. J. Miller and E. R. de los Rios) 1986, Mechanical Engineering Publications, London, pp. 275–293.

ABSTRACT The dependence of $\Delta K_{\rm th}$ on crack size and material properties at a stress ratio R=-1 was studied on various materials and microstructures. The values of $\Delta K_{\rm th}$ of all materials investigated were unified with one geometrical and one material parameter.

The geometrical parameter is the square root of the area determined by projecting defects or cracks onto the plane normal to the maximum tensile stress. The relationship

$$\Delta K_{\rm th} \propto (\sqrt{area})^{1/3}$$

is derived

The most relevant material parameter to unify data is the Vickers hardness and the relationship

$$\Delta K_{\rm th} \propto (H_{\rm v} + {\rm C})$$

is obtained. The constant C reflects the difference of non-propagation behaviour of small cracks in soft and hard metals.

Combining these equations, experiments show that $\Delta K_{\rm th}$ and the fatigue limit $\sigma_{\rm w}$ of cracked members are given by

$$\Delta K_{\text{th}} = 3.3 \times 10^{-3} (H_{\text{v}} + 120) (\sqrt{area})^{1/3}$$

unu

$$\sigma_{\rm w} = 1.43(H_{\rm v} + 120)/(\sqrt{area})^{1/6}$$

Here $\Delta K_{\rm th}$ is in MPa \sqrt{m} , \sqrt{area} in μm , and $\sigma_{\rm w}$ in MPa. These equations are applicable to cracks having \sqrt{area} approximately less than 1000 μm .

Notation

The area which is occupied by projecting a defect or a crack onto the plane normal to the maximum tensile stress α , C, C₁, C₂, n Constants independent of material C', n' Constants dependent on material $H_{\rm B}$ The Brinell hardness number (BHN) $H_{\rm v}$ The Vickers hardness number (DPH) $K_{\rm I}$ Stress intensity factor (mode I) $K_{\rm Imax}$ The maximum value of the stress intensity factor along the front of a three-dimensional crack (mode I)

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THE BEHAVIOUR OF SHORT FATIGUE CRACKS

 ΔK_{th} Threshold stress intensity factor range (R = -1)

 $\Delta K_{\rm eff}$ Effective stress intensity factor range

 $\Delta K_{\rm eff,th}$ Threshold effective stress intensity factor range

l Length of a two-dimensional crack

l_f Fictitious crack length

 l_0 The maximum length of a non-propagating crack observed in

unnotched specimens at the fatigue limit

 ν Poisson's ratio

R Stress ratio

 $\sigma_{\rm U}$ Ultimate tensile strength

 $\sigma_{\rm w}$ Fatigue limit

 σ_{w_0} Fatigue limit of an unnotched specimen

 $\sigma_{\rm Y}$ Yield stress

 σ_0 The maximum tensile stress

Introduction

The threshold stress intensity factor range $\Delta K_{\rm th}$ is required in order to determine the maximum allowable stress when cracks or defects are detected in machine parts and structures under service loading. However, recent experimental studies (1)–(6) show that the value of $\Delta K_{\rm th}$ is dependent on crack size, i.e., the smaller the crack, the smaller the value of $\Delta K_{\rm th}$. On the other hand, the value of $\Delta K_{\rm eff}$ characterizes the growth behaviour of not only a large crack but also a small one, and, accordingly, $\Delta K_{\rm eff,th}$ is almost independent of crack size (7)(8). However, in practice, the application of $\Delta K_{\rm eff}$ or $\Delta K_{\rm eff,th}$ is inconvenient, because we must measure or assume the opening ratio of a crack in real structures, and this measurement is very difficult. Therefore, it is preferable in practice to determine the value of $\Delta K_{\rm th}$ as a function of crack size and geometry and then to estimate the allowable stress from $\Delta K_{\rm th}$ rather than to measure $\Delta K_{\rm eff}$.

The correlation between the fatigue limit $\sigma_{\rm w}$ and the crack length l was first obtained by Frost (9). But at the time when Frost carried out his fatigue tests and obtained the relationship $\sigma_{\rm w}^{\rm n}l=$ constant, the fatigue threshold phenomena and the dependence of $\Delta K_{\rm th}$ on crack length were not recognized.

Kitagawa and Takahashi (3) pointed out that $\Delta K_{\rm th}$ decreases with decreasing crack size. However, an expression for $\Delta K_{\rm th}$ as a function of crack size was not given explicitly. El Haddad *et al.* (10) added the fictitious crack length $l_{\rm f}$ to the real crack length l in order to compensate for the decreasing value of $\Delta K_{\rm th}$ with decreasing crack length. The value of $l_{\rm f}$, considered to be a material constant, was determined from $\Delta K_{\rm th}$ for a large crack, and from the fatigue limit of an unnotched specimen, $\sigma_{\rm w_0}$. However, the physical meaning of $l_{\rm f}$ and the rule for determining $l_{\rm f}$ for various three-dimensional cracks are not clear.

In many previous studies, comparison of $\Delta K_{\rm th}$ values for different materials did not consider the dependence of $\Delta K_{\rm th}$ on crack size and geometry. Such

comparisons are likely to induce erroneous conclusions. One objective of the present paper is to elucidate the dependence of $\Delta K_{\rm th}$ on the shape and size of cracks with special emphasis on small cracks. The large amount of available data on rotating bending fatigue in various materials is analysed. A geometrical parameter \sqrt{area} , which is defined by the square root of the projection area of defects or cracks onto the plane perpendicular to the maximum tensile stress, is proposed in order to unify the effects of various notches, holes, and cracks. An explicit relationship between $\Delta K_{\rm th}$ and \sqrt{area} is confirmed for more than ten materials.

Another objective is to find the most appropriate material parameter which reflects the threshold behaviour. It should be noted that the dependence of $\Delta K_{\rm th}$ on material parameters can be made clear only after the most appropriate geometrical parameter is found. Although various material parameters such as yield stress $(\sigma_{\rm Y})$, ultimate tensile stress $(\sigma_{\rm U})$ and hardness $(H_{\rm v}$ or $H_{\rm B})$, may be correlated with $\Delta K_{\rm th}$, the Vickers hardness number $H_{\rm v}$ was chosen after observing the trend of many data and also for the reasons of simplicity of measurement and availability.

Finally, a simple formula for predicting ΔK_{th} in terms of one material and one geometrical parameter, i.e., H_v and \sqrt{area} , is derived.

A geometrical parameter for small defects or cracks

Effects of small defects, cracks, and inclusions on fatigue strength have been investigated by many researchers (11)-(19). However, their effects are so complicated that no method for unifying them quantitatively has been established. The crucial cause for the absence of a unifying method was an incorrect understanding of the threshold condition for such small defects, cracks, and inclusions. Murakami and Endo (20) showed how to overcome this difficulty by interpreting the fatigue limit not as the critical condition for crack initiation but as the condition for the non-propagation of a crack emanating from defects, cracks, and inclusions; previously Miller et al. (21)(22) had applied a similar consideration to notches. For example, the fatigue limit of a structural component containing a small defect must not be interpreted as a notch problem in which the critical condition of crack initiation is questioned, but should be understood as a problem of a crack which emanates from the defect and stops propagating. Only the interpretation of problems in this manner leads us to find the geometrical parameter for defects, cracks, and even sharp notches. It is reasonable to seek the geometrical parameter from the standpoint that the effects of shapes and sizes of cracks on fatigue strength may be correlated with stress intensity factors, especially with the maximum stress intensity factor along the three-dimensional crack front. Previous studies by Murakami et al. (20)(23)–(31) regarding this problem can be summarized as follows.

First, the stress intensity factors, $K_{\rm I}$, for elliptical cracks in an infinite body under uniform tension were investigated and the approximate relationship

between the maximum value (K_{Imax}) at the tip of the minor axis of an ellipse and the crack area was given as follow (20)

$$K_{\rm Imax} \propto (\sqrt{area})^{1/2}$$
 (1)

Afterwards, three-dimensional stress analyses by the body force method were carried out for surface cracks having various shapes, and the maximum value of the stress intensity factor along their crack front was correlated with the crack areas to give the following equation (23)–(25) (see Appendix 2):

$$K_{\text{Imax}} \cong 0.65 \ \sigma_0 \sqrt{(\pi \sqrt{area})}$$
 (2)

The error in equation (2) may be estimated to be less than 10 per cent. Equation (2) implies that the square root of the crack area projected onto the plane perpendicular to the maximum tensile stress should be adopted as the most relevant geometrical parameter for three-dimensional cracks.

When a specimen has a three-dimensional defect other than a planar crack, the fatigue limit is determined by the threshold condition of the crack emanating from the defect (20)(26)-(30). In this case, the initial three-dimensional shape of the defect is not directly correlated with the stress intensity factor. Rather, the planar domain (area) which is occupied by projecting the defect onto the plane perpendicular to the maximum principal stress should be regarded as the equivalent crack and the stress intensity factor should be evaluated from the equivalent crack, see Fig. 1(b). This may be easily understood from a simple two-dimensional example; the stress intensity factor for an elliptical hole having cracks at the ends of the major axis can be approximately estimated from those of an equivalent crack with a total length of (major axis + crack length). Here, it should be noted that the area of crack emanating from a three-dimensional crack occupies only a small portion of the total projected area (28)(31), see Fig. 1(a), and accordingly the area of an equivalent crack

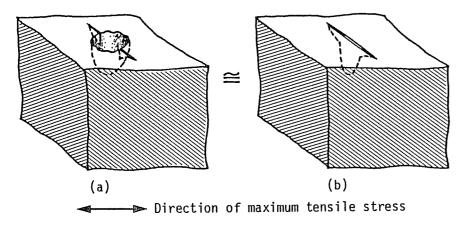


Fig 1 Defect with cracks and its equivalent crack

which should be used for equation (2) may be estimated by the projected area of the initial defect.

From the above discussion, it can be hypothesized that the square root of crack area or the projected area of defects is the relevant geometrical parameter to use when determining quantitatively the fatigue strength of structural components containing various cracks and defects in which no apparent mutual correlations are obvious. With regard to an estimation of \sqrt{area} for irregularly shaped cracks, see Appendix 1.

On the basis of this hypothesis, previous data on rotating bending fatigue were analysed using the parameter \sqrt{area} . The artificial defects investigated in this study are very small drilled holes (20)(28)-(30)(32)-(35) with diameters ranging from 40 μ m to 500 μ m and depths greater than 40 μ m, also very small and shallow notches (35)-(46) with depths ranging from 5 μ m to 300 μ m, and very shallow circumferential cracks (47) with depths greater than 30 μ m, and finally a Vickers hardness indentation (35) of 72 μ m surface length. The shapes of defects and cracks considered are shown in Fig. 2. The effects of work hardening and residual stress by introducing the drilled holes were examined and found to be small (20). In those tests almost all notched specimens were electropolished after introducing the notches (35)-(46) and the cracked specimens were annealed after introducing the fatigue cracks (47). Accordingly the effect of work hardening would be expected to be negligible.

The relationship between $\Delta K_{\rm th}$ and \sqrt{area} is illustrated in Fig. 3. the data in the figure were adopted from various references as well as previous studies by the authors' group. Although this figure may be similar to that of Kitagawa and Takahashi (3), the parameter of abscissa is not crack length but the new parameter \sqrt{area} which can unify the size effect of three-dimensional defects and cracks. Moreover, the adoption of the new parameter \sqrt{area} characterizes the threshold behaviour particularly for the data on very small cracks.

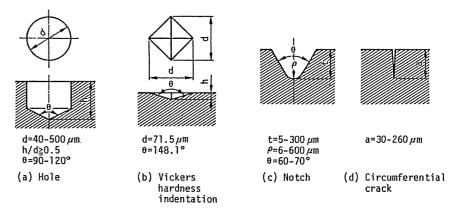


Fig 2 The shapes of defects and cracks investigated in this study



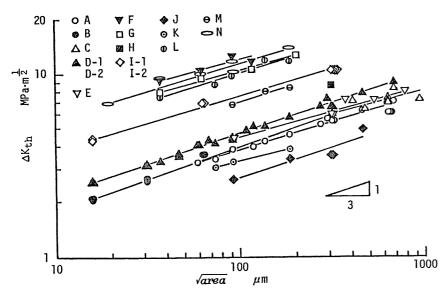


Fig 3 Relationship between ΔK_{th} and \sqrt{area} for various defects and cracks. Letters correspond to the materials given in Table 1

For the region $\sqrt{area} \le 1000 \ \mu m$, the relationship between ΔK_{th} and \sqrt{area} is approximately linear and the following equation holds regardless of material

$$\Delta K_{\rm th} \propto (\sqrt{area})^{1/3}$$
 (3)

Equation (3) characterizes the distinct dependence of $\Delta K_{\rm th}$ on the geometrical parameter \sqrt{area} . Equation (3) implies also that if we investigate $\Delta K_{\rm th}$ of specimens containing a very short two-dimensional crack of length l, we have

$$\Delta K_{\rm th} \propto l^{1/3}$$
 (4)

If we convert the empirical formula obtained by Frost (9), i.e., $\sigma_{\rm w}^3 l = {\rm constant}$, to the same form as equation (4), we have

$$\Delta K_{\rm th} \propto l^{1/6} \tag{5}$$

Equations (4) and (5) contradict each other. This is because the experiments by Frost contain longer or larger cracks than those of the data in Fig. 3. With increasing crack length $\Delta K_{\rm th}$ approaches a constant value depending on the specific material. This indicates that the slope of the relationship between $\Delta K_{\rm th}$ and \sqrt{area} (or l) on a log-log scale varies from one-third to zero with increasing crack size and, therefore, the slope of one-sixth in equation (5) corresponds to a transient value from one-third to zero. This point was recently discussed in detail by Murakami and Matsuda (48).

Vickers hardness as a representative material parameter

Figure 3 presents data on an aluminum alloy and 70/30 brass in addition to various steels. The Vickers hardness $H_{\rm v}$ of these materials is shown in Table 1. It ranges over from 70 to 720 covering a range of a factor of ten.

From the tendency of $\Delta K_{\rm th}$ in Fig. 3, it may be noted that the materials having higher Vickers hardness show higher values of $\Delta K_{\rm th}$ (and accordingly higher fatigue strength). However, the tendency cannot be expressed in a simple form such as $\Delta K_{\rm th} \propto H_{\rm v}$. It has been empirically observed that the fatigue limit of a specimen containing a notch or a defect is not directly proportional to the Vickers hardness. This is presumably because the occurrence of non-propagating cracks may have a different dependency. In other words, a crack is likely to show non-propagating behaviour in soft materials while, on the contrary, it is difficult to find non-propagating cracks at the fatigue limit of hard steels. With increasing hardness non-propagating cracks are experienced only within a narrow range of stress amplitude, and in this case the length of non-propagating cracks is usually very short. Therefore, it may be concluded that $\Delta K_{\rm th}$ is not a function of the form $\Delta K_{\rm th} \propto H_{\rm v}$ or $\Delta K_{\rm th} \propto H_{\rm v}^{\alpha}$. The difference of threshold behaviour between soft and hard materials may rather be expressed by the following formula

$$\Delta K_{\rm th} \propto (H_{\rm v} + {\rm C})$$
 (6)

where C is a constant independent of materials. In order to check this prediction the values of $\Delta K_{\rm th}/(\sqrt{area})^{1/3}$ were plotted against $H_{\rm v}$ from consideration of equation (3) and the validity of equation (6) was confirmed by many data with only a few exceptional data on stainless steels. (See Appendix 3). Now considering equations (3) and (6), the following equation is expected to hold for a wide range of materials

$$\Delta K_{\text{th}} = C_1 (H_v + C_2) (\sqrt{area})^{1/3} \tag{7}$$

where C₁ and C₂ are constants independent of material.

The constants C_1 and C_2 in equation (7) can be determined by the least square method applied to the data in Fig. 3 and we have

$$\Delta K_{\rm th} = 3.3 \times 10^{-3} (H_{\rm v} + 120) (\sqrt{area})^{1/3} \tag{8}$$

where the units of ΔK_{th} are MPa $\sqrt{\text{m}}$ and that of $\sqrt{\text{area}}$ is μ m.

Figure 4 shows the comparison of the experimental data in Fig. 3 and the correlation by equation (8). It is pleasing to note that the various data for $H_{\rm v}$ ranging from 70 to 720 are well represented by equation (8). The reason why equation (8) is not good in predicting $\Delta K_{\rm th}$ for two kinds of stainless steels (34) is presumably because non-propagating cracks are unlikely to be observed in stainless steels even at a sharp notch (49)–(51). The existence of non-propagating cracks in the data of (34) was not checked in the present study.

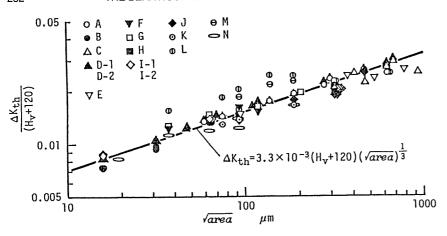


Fig 4 Relationship between $\Delta K_{th}/(H_v+120)$ and \sqrt{area} . Letters correspond to the materials listed in Table 1

Combining equations (8) and (2), the fatigue limit $\sigma_{\rm w}$ of a cracked specimen can be expressed as

$$\sigma_{\rm w} = 1.43(H_{\rm v} + 120)/(\sqrt{area})^{1/6}$$
 (9)

where $\sigma_{\rm w}$ is the nominal stress defined for the gross diameter and has units of MPa.

Murakami and Endo (20) previously proposed an equation of the form $\sigma_w^{n'} \sqrt{area} = C'$ for predicting the fatigue limit of materials containing defects or cracks. Although this equation is very accurate, the negative aspect of it is that we need fatigue tests for individual materials to determine n' and C' (52). This disadvantage is overcome by equations (8) or (9).

Table 1 shows the comparison of the values predicted by equations (8) and (9) with experimental results. For most materials except two kinds of stainless steels, the error is less than 10 per cent.

It should be noted that Fig. 4 and Table 1 include many results of specimens containing an extremely shallow notch (depths ranging from 5 to 20 μ m) or a crack, or very small holes (diameters ranging from 40 to 500 μ m), and although it may be said that the theory of the notch effect has been established for medium or deep notches, conventional theories (22)(43)(44)(53)–(57) may not be applicable to extremely shallow notches. From the viewpoint of the present study however, extremely shallow notches may be placed in the category of small cracks, and so the prediction of the fatigue limit can be simply made by equations (8) or (9).

Table 1 Comparison of predicted values by equations (8) and (9) with experimental results

Material	Defect	H _v	√area (μm)	$\Delta K_{\text{th}} (MPa\sqrt{m})$		$\sigma_w(MPa)$		Error
				Expt	Pred.	Expt	Pred.	(%)
A: S10C (annealed) (36) A: S10C (annealed) (37)	Notch Notch	120 120 120	632 632 632	6.1 6.1 6.1	6.8 6.8 6.8	105 105 105	117 117 117	11. 11. 11.
A: S10C (annealed) (38)	Notch	120 120	316 316	5.5 5.5	5.4 5.4	134 134	132 132	-1. -1.
A: S10C (annealed) (20)(28)	Hole	120 120 120 120 120 120 120 120 120 120	74 60 93 136 119 185 272 298 463 681	3.4 3.2 3.8 4.2 4.0 4.6 5.2 5.7 6.3 7.1	3.3 3.1 3.6 4.1 3.9 4.5 5.1 5.3 6.1	172 181 172 157 157 147 137 142 128	168 174 162 152 155 144 135 133 124	-2. -3. -6. -3. -1. -2. -1. -6. -3.
A: S10C (annealed) (32)	Hole	120	632	7.1	7.0 6.8	118 123	116 117	-1. -4.
B: S30C (annealed) (39)	Notch	153 153 153 153 153 153 153 153 153 153	16 16 16 32 32 32 63 63 63 316 316 316	2.0 2.1 2.1 2.6 2.6 2.7 3.6 3.6 3.6 5.7 5.7	2.3 2.3 2.8 2.8 2.8 3.6 3.6 3.6 6.1 6.1	220 225 225 199 204 208 196 196 196 140 140	247 247 247 220 220 220 196 196 150 150	12. 9. 9. 10. 8. 5. -0.: -0.: 7.: 4.
C: S35C (annealed) (40)	Notch	160 160 160	632 632 632	7.0 7.1 7.3	7.9 7.9 7.9	120 122 126	137 137 137	14. 12. 8.
C: S35C (annealed) (41)	Notch	160 160 160	316 474 949	5.9 6.4 7.2	6.3 7.2 9.1	144 127 101	154 144 128	6.: 13. 26.:
C: S35C (annealed) (27)	Hole	160 160	409 681	7.1 8.2	6.9 8.1	152 137	147 135	-3.2 -1.3
D-1: S45C (annealed) (42)–(44)	Notch	180 180 180 180 180 180 180 180 180	16 16 16 32 32 32 316 316 316	2.6 2.5 2.5 3.2 3.2 3.2 6.6 6.6 6.2	2.5 2.5 2.5 3.1 3.1 6.7 6.7	280 275 275 245 250 245 160 160 151	271 271 271 242 242 242 242 165 165	-3.1 -1.4 -1.4 -1.4 -3.4 -1.4 2.9 2.9 8.9

Continued

Table 1 (Continued)

Material	Defect	H _v	Varea	$\Delta K_{\text{th}} (MPa\sqrt{m})$		$\sigma_w(MPa)$		Error
			(μm)	Expt	Pred.	Expt	Pred.	(%)
D-2: S45C (annealed) (20)(28)	Hole	170	37	3.3	3.2	235	228	-3.2
2 2. 2 . 2 . 4		170	46	3.5	3.4	226	219	-3.0
		170	68	4.3	3.9	226	206	-9.1
		170	48	3.7	3.5	230	218	-5.2
		170	74	4.2	4.0	211	203	-3.9
		170	109	4.8	4.6	201	190 210	-5.5 -7.0
		170	60	4.0	3.7 3.7	226 226	210	-7.0 -7.0
		170	60 93	4.0 4.5	4.3	201	195	-2.8
		170 170	93 93	4.3	4.3	196	195	-0.4
		170	136	5.1	4.9	191	183	-4.1
		170	119	5.1	4.7	201	187	-6.9
		170	185	5.7	5.5	181	174	-3.9
		170	272	6.5	6.2	172	163	-5.2
		170	298	7.2	6.4	181	161	-11.2
		170	463	7.8	7.4	157	149	-4.9
		170	681	8.8	8.4	147	140	-4.7
E: S50C (annealed) (45)	Notch	177	316	5.9	6.7	144	163	13.2 13.2
		177	316	5.9	6.7	144	163	
E: S50C (annealed) (47)	Circum-	177	95	4.4	4.5	196	199	1.5
,	ferential	177	379	7.2	7.1	160	158	-1.2
	crack	177	538	7.1	8.0	133	149	11.9
		177	791	8.0	9.1	123	140	13.5
F: S45C (quenched) (29)(30)	Hole	650	37	9.3	8.5	667	604	-9.4
210.00 (4		650	62	10.3	10.1	568	554	-2.5
		650	93	12.4	11.5	559	518	-7.4
		650	117	11.7	12.4	470	499	6.1
G: S45C (quenched and	Hole	520	37	8.0	7.0	568	502	-11.6
tempered) (29)(30)		520		9.4	8.4	519	460	-11.3
		520		10.5	10.3	421	414	-1.6
		520	202	12.5	12.4	382	378	-1.0
H: S50C (quenched and	Notch	319	316	8.6	9.9	209	241	15.3
tempered) (45)								,
I-1: S50C (quenched and tempered) (45)	Notch	378	316	10.3	11.2	252	273	8.2

Continued

Table 1 (Continued)

Material	Defect	H _v	√area (μm)	$\Delta K_{\rm th} (MPa\sqrt{m})$		$\sigma_w(MPa)$		Error
Material				Expt	Pred.	Expt	Pred.	(%)
I-2: S50C (quenched and	Notch	375	16	4.3	4.1	468	447	-4.4
tempered) (39)		375	16	4.4	4.1	478	447	-6.4
		375	63	6.8	6.5	373	355	-4.8
		375	63	6.8	6.5	373	355	-4.8
		375	316	10.3	11.1	252	272	7.6
		375 375	316 316	10.3 10.3	11.1 11.1	252 252	272 272	7.6 7.6
I: 70/30 brass (46)	Notch	70	316	3.6	4.3	87	104	19.5
170,00 01400 (10)	1,0001	70	316	3.6	4.3	87	104	19.5
1: 70/30 brass (33)	Hole	70	93	2.6	2.8	118	128	8.4
		70	185	3.4	3.6	108	114	5.6
		70	463	4.9	4.8	98	98	-0.3
C: Aluminum alloy	Hole	114	74	3.0	3.2	152	164	7.6
(2017–T4) (33)		114	93	3.3	3.5	147	158	7.2
		114	185	3.9	4.4	123	140	14.1
.: Stainless steel	Hole	355	37	7.4	5.2	530	373	-29.7
(SUS603) (34)		355	74	8.7	6.6	441	332	-24.7
		355	93	9.8	7.1	441	320	-27.5
		355	139	11.7	8.1	432	299	-30.8
		355	185	11.7	8.9	373	285	-23.6
1: Stainless steel	Hole	244	93	6.7	5.4	304	245	-19.4
(YUS170) (34)		244	139	8.0	6.2	294	229	-22.1
		244	185	8.3	6.8	265	218	-17.6
J: Maraging steel (35)	Vickers hardness indenta- tion	720	19	6.9	7.4	686	736	7.3
	Hole	720	37	9.5	9.2	677	659	-2.7
		720	93	11.7	12.5	530	566	6.7
		720	185	13.8	15.8	441	504	14.3
	Notch	720	63	10.0	11.0	546	603	10.4
		720	95	10.2	12.6	454	563	24.0

\$10C: ~0.10% carbon steel \$30C: ~0.30% carbon steel \$35C: ~0.35% carbon steel \$45C: ~0.45% carbon steel \$50C: ~0.50% carbon steel

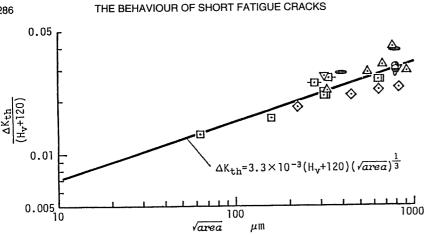


Fig 5 Relationship between $\Delta K_{th}/(H_v + 120)$ and \sqrt{area} . Most of the H_v values are estimated by equation (10). See Table 2 for symbol references

Applications

After obtaining equation (8), the data of other references (9)(47)(57)(58)-(64) were investigated. Few of them present the hardness of materials. Therefore, $H_{\rm v}$ was estimated by using the relationship between $H_{\rm v}$ and $H_{\rm B}$ (ASTM E-48-43-T) and an empirical formula which is thought to hold between ultimate tensile strength and $H_{\rm B}$, i.e.

$$\sigma_{\rm U} \cong 0.36(9.8 \times H_{\rm B}) \tag{10}$$

Figure 5 and Table 2 provide details of the comparison between the prediction by equation (8) and the experimental data. Although these references lack the data for very small values of \sqrt{area} , it may be concluded that equation (8) predicts $\Delta K_{\rm th}$ very well for cracked or notched specimens having \sqrt{area} less than 1000 μ m.

Table 2 A collection of data from the different studies used in Fig. 5

Material	Defect	Researchers			
① 0.12% C steel ② 0.53% C steel	Fatigue crack	Isibasi and Uryu (58)			
△ SF60	Artificial crack	Ouchida and Kusumoto (59)			
	Fatigue crack	Frost (9)			
□ S20C	Circumferential notch	Nisitani (57)(61)			
-⊡- S20C	Drilled hole	Nisitani and Kage (60), Hayashi et al. (62)			
0 S25C	Fatigue crack	Awatani and Matsunami (63)			
S 35C	Fatigue crack	Kobayashi and Nakazawa (64)			
 Eutectoid steel 	Circumferential notch	Kobayashi and Nakazawa (47)			

The limitation of the applicability of equations (8) and (9)

As seen in Fig. 4, equations (8) and (9) can be applied to small defects or cracks within some range of the value of \sqrt{area} . Although the upper limit of the size of \sqrt{area} for the application of these equations is uncertain at present, it may be approximately 1000 μ m. The lower limit is dependent on material properties and microstructures. In experiments we have a finite value of the fatigue limit $\sigma_{\rm w}$ for specimens containing no defects or cracks. Theoretically, in this case, $\sqrt{area} = 0$ and accordingly $\sigma_{\rm w} = \infty$. But this never occurs, because cracks along slip bands or grain boundaries nucleate as a result of reversed slip in grains, that is, \sqrt{area} is not zero, and accordingly the fatigue limit of defect-free specimens σ_{w_0} becomes finite. Therefore, as discussed in previous studies (28)–(30), the lower limit of \sqrt{area} for the application of equations (8) and (9) is related to the maximum length, l_0 , of non-propagating cracks observed in unnotched (defect-free) specimens. It follows that even if a specimen contains small defects or cracks before fatigue tests, and if the fatigue limit σ_w predicted from the value of \sqrt{area} and equation (9) is greater than σ_{w_n} , the value of σ_{w} is never measured because such defects do not lower the fatigue strength of the specimen and they are virtually harmless. When we know the value of σ_{w_0} in advance, the lower limit of \sqrt{area} can be determined from equation (9). When $\sigma_{\rm w_a}$ is unknown, its approximate value can be estimated by the empirical formula

$$\sigma_{\mathbf{w}_{0}} \cong 0.5\sigma_{\mathbf{U}} \cong 1.6H_{\mathbf{v}} \tag{11}$$

where σ_U is the ultimate tensile strength in MPa. So far, it has been said that equation (11) is not necessarily applicable to high-strength or hard steels (65). However, such a conclusion was based on experiments in which the original site of fatigue fracture - whether it was slip bands or defects - was not identified exactly. Murakami and Endo (29)(30) showed on the basis of careful investigations of fracture origins that equation (11) is applicable to hard steels when defects are not the cause of fatigue fracture.

Concluding remarks

The dependence of ΔK_{th} on crack size and material properties was investigated on more than ten materials and microstructures. The value of ΔK_{th} of all materials were unified with one geometrical together with one material parameter.

The geometrical parameter is the square root of the area which is occupied by projecting the defects or cracks onto the plane normal to the maximum tensile stress. It was found that

$$\Delta K_{\rm th} \propto (\sqrt{area})^{1/3}$$

The material parameter to unify data is the Vickers hardness value, H_v . The

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influence of microstructural variables and the difference of materials can be unified by the following equation

$$\Delta K_{\rm th} \propto (H_{\rm v} + 120)$$

The constant of 120 in the above equation reflects the experimental fact that relatively large non-propagating cracks are likely to be observed in low strength metals in comparison with hard metals. Combining the above two equations, we find by experiment that

$$\Delta K_{\text{th}} = 3.3 \times 10^{-3} (H_{\text{v}} + 120) (\sqrt{area})^{1/3}$$

for threshold stress intensity factor ranges at a stress ratio, R, equal to -1, and

$$\sigma_{\rm w} = 1.43(H_{\rm v} + 120)/(\sqrt{area})^{1/6}$$

for the fatigue limit, where the units of the quantities in these equations are MPa \sqrt{m} for ΔK_{th} , MPa for σ_{w} , and μm for \sqrt{area} .

Although the upper limit of \sqrt{area} for the application of the above equations is uncertain at present, it may be approximately $1000~\mu m$. The lower limit can be estimated from the fatigue strength of unnotched (defect-free) specimens or from the length of the maximum non-propagating crack which is observed at the fatigue limit of an unnotched specimen.

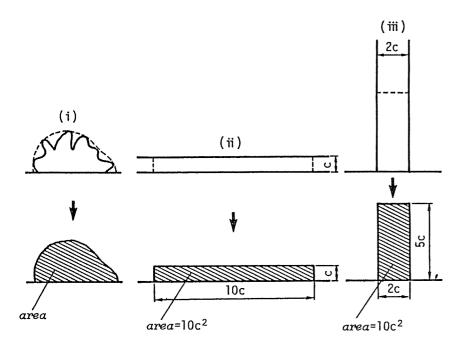


Fig 6

Appendices

Appendix 1

Figure 6 gives rules for the estimation of \sqrt{area} for irregularly shaped cracks and very slender cracks (24) (25). (See also (20) (66).)

The long shallow and very deep cracks shown in Fig. 6 as (ii) and (iii), respectively, must be bounded by a maximum length of 10c and 5c, respectively, for the estimation of effective crack area.

Appendix 2

Figure 7 shows the relationship between the maximum stress intensity factor K_{Imax} and \sqrt{area} for surface cracks (elastic analysis) (24)(25). (See also (20)(66).)

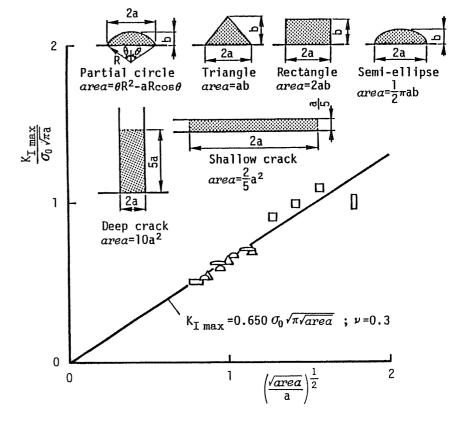
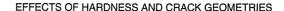


Fig 7



 $\frac{3}{8}$ $\frac{2}{4}$ $\frac{1}{4}$ $\frac{3}{4}$ $\frac{3}$

Fig 8

Appendix 3

Figure 8 shows the dependence of $\Delta K_{\rm th}/(\sqrt{area})^{1/3}$ on the material parameter $(H_{\rm v})$ for various materials.

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