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This paper provides a reliability analysis of one of the most critical power plant component - welded steamline, taking into account their exposure to the static pressure, low-cycle fatigue and creep damage.

### INTRODUCTION

Most of the power plant users run their equipment at high professional level, but they rarely pay enough attention to the weldments unless a failure occurs. It is typical that only then a serious monitoring of weldments behavior starts. Therefore, it is the aim of this paper to provide a reliability analysis of one of the most critical power plant component - welded steamline, having in mind their exposure to the static pressure, low-cycle fatigue and creep damage.

### ESTIMATION OF OPERATION CAPACITY REDUCTION

In order to make a probabilistic reliability analysis of welded steamline containing a defect in welded joint, the experimental data should be statistically worked out, using certain probabilistic level  $\alpha$  (1,2):

$$P(y - \sigma_{y/x} t_{\alpha, k} < a_y < y + \sigma_{y/x} t_{\alpha, k}) = \alpha \quad (1)$$

$$y = a + bx = \bar{y} + r \frac{\sigma_y}{\sigma_x} (x - \bar{x}) ; \quad \sigma_{y/x} = \sigma_y (1 - r^2)^{\frac{n-1}{n-2}}$$

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where  $\bar{x}$  and  $\bar{y}$  denote the mean values of  $x$  and  $y$ ,  $r$  - correlation coefficient,  $\sigma_x$  and  $\sigma_y$  - mean quadratic deviation values of  $x$  and  $y$  and  $t_{\alpha, k}$  - Student's coefficient.

As an example steamline (inner diameter 219 mm, wall thickness 24 mm) with weld bridge seam - poor penetration in weldment root in the form of crack-like defect ( $a_0 = 2$  mm deep,  $2l_0 = 5$  mm long).

Steamline was made of 14 MoV 63 steel and loaded as follows: pressure 14 MPa, temperature 520°C, stress amplitude 98 MPa. Allowed design stress at 520°C for this steel is 93 MPa. Designed number of start-ups and shut-downs is  $N=5 \cdot 10^4$ .

Weldment strength dependence on a defect size (represented by a general parameter  $T$ ) for the static loading has been found experimentally (3):

$$R_{zs} = 400 - 16.5 \cdot T \quad (2)$$

where  $T$  can be expressed as follows (for both static and cyclic loads):

$$T = \frac{1}{\pi} \left( \frac{K}{\sigma} \right)^2 = \frac{a}{Q} \left[ 1 + 0.12 \left( 1 - \frac{a}{l} \right) \right]^2 \left( \frac{2s}{\pi a} \operatorname{tg} \frac{\pi a}{2s} \right) \quad (3)$$

where  $a$  denotes defect depth (mm),  $l$  - defect length (mm),  $Q$  - defect shape coefficient and  $s$  - weldment thickness. For given data,  $a_0 = 2$  mm,  $2l_0 = 5$  mm, one gets  $T_0 = 1.03$  mm.

The corresponding value of stress intensity factor is:

$$K_u = \sigma_{\max} \sqrt{\pi \cdot T_0} = 3.87 \text{ MPa}\sqrt{\text{m}} \quad (4)$$

Figure 1 shows probabilistic dependence of weldment strength on a general crack size parameter  $T$ . Crossing of the regression line and allowed design stress defines the upper limit of general crack size parameter  $T_n$ . Thereby, probability of residual strength being equal or greater than the allowed stress ( $\sigma_d$ ) is determined by the marked area,  $\beta_n$ .

Therefore,  $T_n$  can be calculated as follows:  $T_n = (a - \sigma_d)/b = 18.6$  mm. Now, the admissible value of a general crack size parameter,  $T_n$ , can be obtained for the 95% statistical reliability:

$$T_d = T_n - Z_{0.95} \sigma_x / \sqrt{n} = 16.5 \quad (5)$$

where  $Z_{0.95} = 1.96$  is the argument value for the 95% statistical reliability,  $\sigma_x = 4.76$ ,  $n = 20$ , as defined in (3), under the probability  $\beta_n$  that residual strength will not be less than  $\sigma_d$ .

Required number of cycles to initiate the crack from the defect poor root penetration type is given by (3):

$$N_i = 95.6 - 15.7 \cdot K_u$$

i.e., with 95% statistical reliability:

$$N_i = 95.6 - 15.7 \cdot K_u - \sigma_{y/x} \sqrt{\frac{1}{n} + \frac{(x - \bar{x})^2}{(n-1)\sigma_x^2}} t_{\alpha, k} = 29,400$$

where  $\sigma_x = 2.62$ ,  $n = 31$  and  $\sigma_{y/x} = 21.3$ , (3). Number of cycles needed for the fatigue crack growth from  $T_0$  to  $T_k$  is:

$$N_r = N - N_i = 50,000 - 29,400 = 20,600$$

Having in mind the Paris law, one can write:

$$N_f = \int_{T_0}^{T_k} \frac{dT}{C(\Delta K)^m} = \frac{2 \cdot 10^6 (K^*)^m}{(m-2)\sigma_{max}^m T^{(m-2)/2}} \Big|_{T_0}^{T_k}$$

where  $\frac{dT}{dN} = 10^{-6} \left( \frac{K}{8.6} \right)^{3.5}$ ,  $m = 3.4$ ,  $K^* = 8.5$ , (3), giving  $T_k = 17.5$  mm for the final value of general crack size parameter.

Therefore, the appropriate argument value  $Z_k$  is

$$Z_k = \frac{T_n - T_k}{\sigma_x} \sqrt{n} = 1.033$$

giving the probability for steamline failure

$$Y_k = 1 - P_k = 14.4\%$$

where  $P_k = 85.6\%$  is obtained directly from  $Z_k$ .

#### CONCLUSIONS

It has been shown, with 95% statistical confidence, that due to defect of poor root penetration type in weldment, (2 mm deep and 5 mm long) steamline operation capacity is reduced for 14.4% after 50,000 load cycles, out of which 29,400 cycles were needed for the crack initiation.

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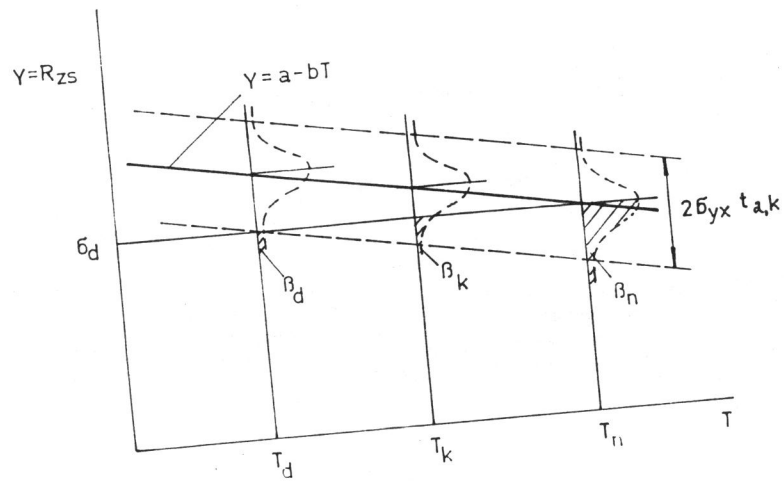


Figure 1 Probabilistic diagram of weldment strength dependency on the general parameter of defect size