CALCULATION OF STRESS INTENSITY FACTORS FOR COMBINED MODE BEND SPECIMENS

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#### INTRODUCTION

In order to test combined mode fracture criteria experimentally it is necessary to use specimens with a wide range of K<sub>I</sub> and K<sub>II</sub> and to obtain calibrated curves of K<sub>I</sub> and K<sub>II</sub> values for these specimens either by calculation or by experiment. As no such calculated curves of  $K_{
m I}$  and  $K_{
m II}$ values were readily available for three-point-bend specimens with cracks in an unsymmetrical position (Figure 1), such curves were obtained by use of the boundary collocation method and the finite element method. The boundary collocation method was first used by Gross et al [1,2,3], to calculate K<sub>I</sub> values for opening mode specimens. Later the method was used to calculate K<sub>I</sub> and K<sub>II</sub> values for some combined mode specimens [4,5]. As for the finite element method used to determine stress intensity factors, growing interest is now directed to special elements at the crack tip [6,7]. A term in  $\sqrt{r}$  is included in the displacement functions of these special elements, where r is the distance from the given point to the crack tip. As this term gives the required singularities of stresses and strains in the vicinity of the crack tip, a higher accuracy can often be obtained with relatively fewer elements. Since the other elements around the special elements are still ordinary ones, whose displacement functions do not contain the terms in  $\sqrt{r}$ , the results obtained by use of these special elements are not very satisfactory, especially in the case when the size of the elements is decreasing. It should be noted that convergence of the results cannot be insured for elements of diminishing size, if the condition of constant strain is not satisfied by those special elements [8], as is the case with commonly adopted ones. However, the distorted isoparametric elements, proposed by Henshell et al [9] and Barsoum [10], satisfy the constant strain condition and their displacement functions contain the terms in √r. The 8-noded isoparametric quadratic and triangular elements with the mid-side nodes near the crack tip at the quarter point have been used in the vicinity of the crack tip and their displacement functions contain the terms in  $\sqrt{r}$ . Now we have succeeded in including terms in  $\sqrt{r}$  in the displacement functions of any isoparametric elements at any arbitrary positions. When these special elements are used in a wider area, not restricted to the vicinity of the crack tip, the accuracy of the calculated results has improved considerably.

Analysis of the energy-momentum tensor, which was proposed by Eshelby [11] and was later used in combined mode fracture criteria by Hellen et al [12], showed that this kind of application is questionable theoretically, and that the results thus obtained are doubtful [13]. In the meantime, an approximate relation between  $K_{\rm I}$  and  $K_{\rm II}$  is used to derive the approximate

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formula for  $K_{\mbox{\scriptsize II}}$  for the bend specimens mentioned above. The results calculated by this formula are compared with those from the boundary collocation method.

## 2. BOUNDARY COLLOCATION METHOD

Consider the following expansion of the stress function with the crack tip as the centre:

$$X = \sum_{j=1}^{\infty} r^{\frac{j}{2} + 1} \left\{ C_{j} \left[ -\cos\left(\frac{j}{2} - 1\right)\theta + \frac{\frac{j}{2} + (-1)^{j}}{\frac{j}{2} + 1} \cos\left(\frac{j}{2} + 1\right)\theta \right] + D_{j}(-1)^{j+1} \left[ \sin\left(\frac{j}{2} - 1\right)\theta - \frac{\frac{j}{2} + (-1)^{j+1}}{\frac{j}{2} + 1} \sin\left(\frac{j}{2} + 1\right)\theta \right] \right\}.$$
 (1)

According to the approach adopted by Gross et al [1,2,3], the expansion is truncated to the first 2N terms and M(M>N) points on the boundary of the specimen are chosen. From the boundary conditions on these M points, 2M equations are obtained and the 2N coefficients in the truncated expansion can be determined. The values of  $K_{\rm I}$  and  $K_{\rm II}$  are determined from the first two coefficients:

$$K_{I} = -C_{1}\sqrt{2\pi}$$
,  $K_{II} = D_{1}\sqrt{2\pi}$ . (2)

Note that as the term in  $\mathrm{D}_2$  in equation (1) is identical with zero, this term should be deleted from the resulting simultaneous equations, otherwise an overflow will take place during the calculation if M is equal to N. The overflow was mentioned in [4] by Wilson et al, and the reason is now explained here.

We take 43 terms and choose 63 collocation points. The calculated results are shown in Table 1. It can be seen from Table 1 that  $K_{\rm I}BW^{3/2}/M$  and  $K_{\rm II}BW^{1/2}/Q$  depend on a/W only in a wide range (as long as the crack tip is not very close to the concentrated forces and the support points). It follows that the values of  $K_{\rm I}$  and  $K_{\rm II}$  can be determined approximately from the bending moment and the shearing force on the crack section, respectively.

## 3. FINITE ELEMENT METHOD

An 8-noded isoparametric quadratic element is shown in Figure 2. Its shape functions are

$$N_{\mathbf{i}} = \frac{1}{4} \left( 1 + \xi_{\mathbf{i}} \xi \right) \left( 1 + \eta_{\mathbf{i}} \eta \right) \left( \xi_{\mathbf{i}} \xi + \eta_{\mathbf{i}} \eta - 1 \right)$$

for the corner nodes,

$$N_{\dot{1}} = \frac{1}{2} (1 - \xi^2) (1 + \eta_{\dot{1}} \eta)$$
 (3)

for the mid-side nodes with  $\xi_i$  = 0, and

$$N_i = \frac{1}{2} (1 + \xi_i \xi) (1 - \eta^2)$$

for the mid-size nodes with  $\eta_1$  = 0. Taking the side  $\eta$  = +1, we have (Figure 3)

$$N_{1} = -\frac{\xi(1-\xi)}{2}$$

$$N_{2} = 1 - \xi^{2}$$

$$N_{3} = \frac{\xi(1+\xi)}{2}$$
(4)

By the coordinate transformation used for the isoparametric elements, this side is assumed to be mapped into a segment AB on a line passing through the crack tip O (Figure 3). The lengths of OA and AB are equal to  $L_0$  and  $L_0$ , respectively. The point,  $\xi=0$ , is mapped into a point C, which is supposed to divide the segment AB into a ratio of p and (1-p). If the coordinate on the segment AB, after the transformation, is denoted by x, it follows that

$$x = -\frac{\xi(1-\xi)}{2} L_0 + (1-\xi^2)(L_0 + pL) + \frac{\xi(1+\xi)}{2}(L_0 + L).$$
 (5)

Let  $L_0/L = k$ , it can be shown that

$$\xi = -\left[1 + 2k + \sqrt{4k(k+1)}\right] + 2(\sqrt{1+k} + \sqrt{k})\sqrt{\frac{x}{L}}$$
 (6)

when

$$p = \frac{1}{4} \left[ \sqrt{4k(k+1)} + 1 - 2k \right] . \tag{7}$$

By substituting equation (6) into the relevant formulae of the isoparametric element, we obtain expressions for the displacement that include terms in  $\sqrt{r}$ . Let k=0, then the relations given in [9] and [10] can be obtained. From equation (7), it is easily seen that p approaches 1/2 as k is getting larger. That is to say, that the distorted elements approach the normal (undistorted) ones farther away from the crack tip.

For the 12-noded isoparametric quadratic element (see [8]), if we assume that the mid-side nodes of the distorted elements divide the side into a ratio of p, (q-p) and (1-q) (Figure 4), it can be shown that equation (6) is again established, when

$$p = \frac{1}{9} [1 - 4k + 4 \sqrt{k(k+1)}]$$

$$q = \frac{1}{9} [4 - 4k + 4 \sqrt{k(k+1)}].$$
(8)

The corresponding expressions for the displacement thus obtained contain terms in  $\sqrt{r}$  and  $r\sqrt{r}$  .

To test the method we consider a three-point-bend specimen with a crack at a symmetrical position. The geometry of the specimen and its finite element idealization are shown in Figure 5. In the vicinity of the crack tip we use triangular elements, which were shown to be superior to the quadratic ones [10].

First we use the same procedure as given in [10]. Only those triangular elements in the vicinity of the crack tip are taken to be distorted ones, with the mid-side nodes near the crack tip at the quarter point and all other elements taken to be normal ones. The final results are shown in Figure 6a. By the use of the calculated values of the displacements of

the points on the crack edges, the apparent values of the stress intensity factors can be determined from

$$\bar{K}_{I} = \frac{E}{4(1 - v^{2})} \sqrt{\frac{2\pi}{r}} u$$
(9)

 $ar{K}_I$  is plotted against distance, r. By analysing the expansion of the displacement at the crack tip, it can be shown that the apparent value of  $ar{K}_I$  is a linear function of r, if r is sufficiently small. The intersecting point of the straight part of the curve on the vertical axis (r = 0) gives the true  $K_I$  value. Some points near the crack tip that deviate from the straight line can be seen in Figure 6a. This indicates that these apparent values of  $K_I$  are questionable and should be discarded.

Secondly, we re-calculate the mid-side nodes according to equation (7) for all elements in the shaded area of Figure 5. The final results are shown in Figure 6b in the sense that all points near the crack tip fall on a straight line, as expected from the analysis. The intersecting point gives  $K_{\rm I}BW^{3/2}/M = 7.79$ , with a/W = 0.4. This is in good agreement with the result calculated by the boundary collocation method:  $K_{\rm I}BW^{3/2}/M = 7.71$ .

# 4. APPROXIMATE RELATION BETWEEN K, AND K, II

For any plane configuration with a crack as shown in Figure 7 it can be proved that

$$J_1 = -\frac{\partial U}{\partial I} = \int_C W dy - \underline{T} \cdot \frac{\partial U}{\partial x} ds$$
 (10)

$$J_2 = -\frac{\partial U}{\partial s} = \int_C -W dx - \underline{T} \cdot \frac{\partial \underline{u}}{\partial y} ds$$
 (11)

where U is the total potential energy of the system and C is the exterior contour of the configuration. Equation (11) gives the rate of the increase of the total potential energy as the crack translates in the direction perpendicular to the crack. Equation (10) defines the J-integral, whose value is path-independent, as long as the path starts at the lower edge of the crack and ends at the upper edge. As for equation (11), it can be proved that the value of the integral is also path-independent, if the points on the crack edges remain intact [11]. If a contour D sufficiently near the crack tip is taken, it can be shown that

$$J_{1}^{*} = f_{D} \operatorname{Wdy} - \underline{T} \cdot \frac{\partial \underline{u}}{\partial x} ds = \frac{(1 + v)(1 + \kappa)}{4E} (K_{1}^{2} + K_{11}^{2})$$

$$J_{2}^{*} = f_{D} - \operatorname{Wdx} - \underline{T} \cdot \frac{\partial \underline{u}}{\partial y} ds = -\frac{(1 + v)(1 + \kappa)}{4E} \cdot 2K_{1}K_{11}$$
(12)

where

$$\kappa = \begin{cases} \frac{3 - v}{1 + v} & \text{for plane stress} \\ 3 - 4v & \text{for plane strain.} \end{cases}$$
 (13)

Due to the properties of the J-integral,  $J_1$  is equal to  $J_1^{\, \prime}.$  It can be shown that

$$R = J_{2}^{\dagger} - J_{2} = \int_{\mathbf{r}_{1} + \mathbf{r}_{2}} W dx = \frac{(1 + \nu)(1 + \kappa)}{8E} \int_{\mathbf{r}_{1} + \mathbf{r}_{2}} \sigma_{X}^{2} dx$$
 (14)

where  $\Gamma_1$  and  $\Gamma_2$  are the upper and lower edges of the crack, respectively. For the upper edge the integration proceeds from left to right and for lower edge from right to left. Since  $\sigma_X^2$  takes the same value on the upper and lower edges in the vicinity of the crack tip and  $\sigma_X$  dwindles when the point moves towards the open end of the crack, it is expected that R will be a small quantity and will not be very sensitive to a small change in the crack length. So it is reasonable to assume that

$$\frac{\partial R}{\partial I} \approx 0$$
 , (15)

Combining equations (15), (10), (11) and (12), we obtain

$$\frac{\partial J_1'}{\partial s} \approx \frac{\partial J_2'}{\partial 1} \,, \tag{16}$$

from which we obtain the following approximate relation between  $K_{\rm I}$  and  $K_{\rm II}$ :

$$K_{II} \frac{\partial K_{I}}{\partial I} + K_{I} \frac{\partial K_{II}}{\partial I} + K_{I} \frac{\partial K_{I}}{\partial S} + K_{II} \frac{\partial K_{II}}{\partial S} = 0 . \tag{17}$$

If we further assume that  $\textbf{K}_{\text{I}}$  and  $\textbf{K}_{\text{II}}$  can be determined by the bending moment M and the shearing force Q on the crack section, respectively, we can write

$$K_{I} = \frac{M}{BW^{3/2}} f_{b} \left(\frac{a}{W}\right) \tag{18}$$

$$K_{II} = \frac{Q}{BW^{1/2}} f_s \left(\frac{a}{W}\right)$$
.

After substituting equation (18) into equation (17), we get the following equation:

$$f_{b}\left(\frac{a}{W}\right) \frac{df_{s}\left(\frac{a}{W}\right)}{d\left(\frac{a}{W}\right)} + \frac{df_{b}\left(\frac{a}{W}\right)}{d\left(\frac{a}{W}\right)} f_{s}\left(\frac{a}{W}\right) - \left[f_{b}\left(\frac{a}{W}\right)\right]^{2} = 0 \quad . \tag{19}$$

The equation is solved to obtain

$$f_{s}\left(\frac{a}{W}\right) = \frac{1}{f_{b}\left(\frac{a}{W}\right)} \int_{0}^{a/W} \left[f_{b}\left(\frac{a}{W}\right)\right]^{2} d\left(\frac{a}{W}\right) . \tag{20}$$

For  $f_b(a/W)$ , we make use of the results for pure bending due to Benthem et al [14], the calculated values of  $f_s(a/W)$  according to equation (20) are given by Table 2 and are in reasonably good agreement with the results calculated by the boundary collocation method.

### CONCLUDING REMARKS

The paper has outlined the results of three methods used in the calculation of  $\textbf{K}_{I}$  and  $\textbf{K}_{II}$  for combined mode bend specimens. If an estimate is to be

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### Workshop

Table 1 Calculated results for  $K_1^\star$  and  $K_{11}^\star$  for three-point-bend specimens with s/W=4 by the boundary collocation method

a/W	2s <sub>1</sub> /s	0	1/6	2/6	3/6	4/6	5/6	11/12
0.4	κ <sub>1</sub> κ <sub>1</sub>	7.71 0	8.50 1.032	8.55 1.400	8.36 1.350	8.33 1.298	8.50 1.376	8.50 1.644
0.45	K* K*II	8.86 0	9.67 1.142	9.72 1.562	9.38 1.488	9.48 1.466	9.55 1.464	
0.50	K* K* II	10.27 0	11.48 1.410	11.50 1.864	11.60 1.840	11.15	11.59	11.53 1.760
0.55	K* K*1 I I	12.11 0	13.30	13.60	13.03 2.050	12.90 1.976	13.46 2.100	***************************************
0.60	K* K*1	14.47 0	14.25 2.348	14.65 2.248	14.91 2.276	14.88 2.320	14.74	14.50 2.090

$$K_{I}^{*} = K_{I}BW^{3/2}/M$$
  $K_{II}^{*} = K_{II}BW^{1/2}/Q$ 

a/W	f <sub>b</sub> (a/W)	f <sub>s</sub> (a/W)	f;(a/W)	Difference in percentage
0.05	2.54	0.0636		
0.10	3.51	0.180		27
0.15	4.26	0.327		7
0.20	4.97	0.496		
0.25	5.67	0.667		
0.30	6.45	0.857		
0.35	7.32	1.080		20
0.40	8.35	1.317	1.350	-2.5
0.45	9.60	1.557	1.488	4.4
0.50	11.12	1.838	1.840	-0.1
0.55	13.09	2.125	2.050	3.5
0.60	15.66	2.441	2.276	6.8
0.65	19.17	2.794		
0.70	24.15	3.077	,	

Note:  $f_b(a/W)$  and  $f_s(a/W)$  are identical with  $K_1^*$  and  $K_{II}^*$  in Table 1.  $f_s^*(a/W)$  is calculated by the boundary collocation method for the case  $2s_1/s = 3/6$ .

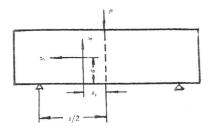


Figure 1 Three-point-bend specimen with crack in unsymmetrical position

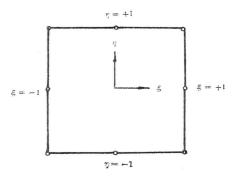


Figure 2 8-noded isoparametric quadratic element

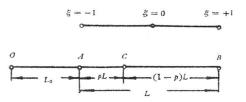


Figure 3 Distorted side of 8-noded isoparametric element

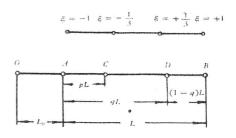


Figure 4 Distorted side of 12-noded isoparametric element

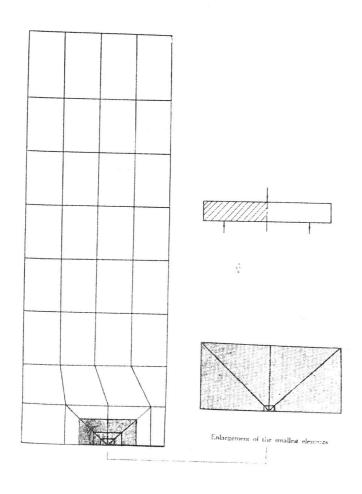


Figure 5 Finite element idealization of a three-point-bend specimen with a crack in a symmetrical position, a/W = 0.4

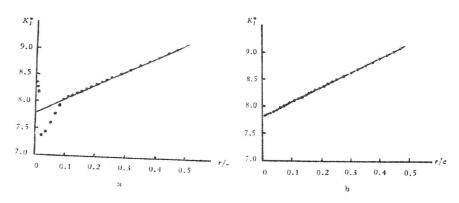


Figure 6 Apparent values of stress intensity factors for specimen shown in Figure 5

- a. Special elements are restricted in vicinity of crack tip
- b. Special elements are not restricted in vicinity of crack tip.

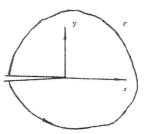


Figure 7 Plane configuration with crack