# PROBABILISTIC STUDY ON FATIGUE LIFE OF PROOF TESTED CERAMICS SPRING

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A new method was proposed to evaluate the probabilistic distribution of residual flaw size after proof test. The theory based on both process zone size failure criterion and two parameter Weibull distribution of  $\sigma_{\rm f}$  and  $K_{\rm LC}$ . A numerical crack growth analysis and fatigue test on  $Si_3N_4$  model spring were conducted to predict the fatigue life. The calculated result showed good agreement with the experimental probabilistic fatigue life.

#### INTRODUCTION

Ceramics has excellent resistivities to heat, corrosion and wear, and ceramics coil spring has been developed[1]. However, generally speaking, ceramics is less reliable compared with metals. Because, their fracture toughness are not so high and they are sensitive to flaws. Their allowable flaw size is so small that it is almost impossible to detect flaws by NDI and repair them. To overcome this problem with reality, proof test is developed. Proof test is very useful for static load and it is verified. However, it is useful or not to cyclic load is not well verified. Because fatigue life dominantly depends on initial flaw size. Then it is very important to determine the distribution of residual flaw size after proof test. However, most structural ceramics show non-linear fracture behaviour[2]. Then, to determine the distribution of residual flaw size, non-linear fracture criterion[3] should be used. In this paper, new theory based on process zone size failure criterion[3] is proposed, to determine the residual flaw size distribution after proof test. Fatigue test has been made on proof tested ceramics model spring, and it is verified that the theory is useful to evaluate the probabilistic fatigue life.

#### THEORY

## Correlation between Proof Stress and Probability of Residual Crack Size

Fig.1 shows a correlation between fracture stress(  $\sigma_{\rm C}$  ) and equivalent crack length(a<sub>e</sub>) in structural ceramics. Solid line shows a average correlation between  $~\sigma_{\rm C}$  and a<sub>e</sub>. This

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line is given by using process zone size failure criterion for ceramics[3]. It is already reported that this criterion showed very good agreement with experimental results for many kinds of structural ceramics[3].

$$\frac{\pi}{8} \left| \frac{\overline{K_{1C}}}{\sigma_{f}} \right|^{2} = a_{e} \left| s e c \right| \left| \frac{\pi \sigma_{c}}{2 \sigma_{f}} \right| = 1$$
(1)

where,  $\overline{\sigma_f}$  is average fracture stress of plain specimen and  $\overline{K_{fc}}$  is average plane strain fracture toughness.

By using Eq(1), average residual crack size  $a_p$  after proof stress of  $\sigma_P$  is given.

$$a_{p} = \frac{\pi}{8} \left[ \frac{\overline{K}_{TC}}{\overline{\sigma}_{f}} \right]^{2} \left[ s e c \left[ \frac{\pi \sigma_{p}}{2 \overline{\sigma}_{f}} \right] - 1 \right]^{-1}$$
 (2)

If  $\sigma_f$  and  $K_{1C}$  are constant, the residual crack size  $a_P$  is a constant and given by Eq(2). However, both  $\sigma_f$  and  $K_{1C}$  show scatter[3], generally, then the residual crack size is not constant but probabilistic one. Now we assume that the distribution of both  $\sigma_f$  and  $K_{1C}$  are given by 2-parameter Weibull distribution function. Firstly, we assume the case where fracture stress of plain specimen  $\sigma_f$  is the constant value of  $\sigma_0$  and only  $K_{1C}$  is probabilistic one as shown in Fig.1. The probability  $H(a_{PX})$  is defined that larger crack  $a_{PX}$  than  $a_P$  will reside. The probability  $H(a_{PX})$  is equal to the probability that  $K_{1C}$  is greater than  $K_P$  as shown in Fig.1 by alternate long and short dash line and chain line. Then it can be given by following equation, easily.

$$H(a_{px}) = 1 - F(K_p)$$
 (3)

where, F(K) is two parameter Weibull distribution function of  $K_{1C}$ , and  $K_p$  is easily given from Eq(1) by substituting  $a_p$ ,  $\sigma_p$ ,  $\sigma_n$  and  $K_p$  for  $a_e$ ,  $\sigma_e$ ,  $\sigma_f$  and  $K_{1C}$ , respectively.

Generally, both  $\sigma_f$  and  $K_{\rm IC}$  are probabilistic. Then, probability  $dG(a_{\rm PN})$  is defined that larger crack  $a_{\rm PN}$  than  $a_{\rm P}$  will reside for the range from  $\sigma_0$  to  $\sigma_0$  +d  $\sigma_0$ . The probability  $dG(a_{\rm PN})$  is given by using probabilistic density function  $f(\sigma_0)$  of  $\sigma_0$ , easily.

$$d G(a_{px}) = H(a_{px}) f(\sigma_0) d(\sigma_0)$$
(4)

where  $f(\sigma_0)$  was given by substituting  $\sigma_0$  for  $\sigma_f$  of  $f(\sigma_f)$ .

Then the probability  $G(a_{PN})$  where larger crack  $a_{PN}$  than  $a_P$  will reside for the proof stress  $\sigma_p$  is given by integrating Eq(4) from  $\sigma_p$  to infinite. Subsequently, fatigue life reliability  $R(a_{PN})$  is given by Eq(5) as a function of proof stress  $\sigma_p$ 

$$R(a_{px}) = \{1 - G(a_{px})\} \times 1 \ 0 \ 0 \%$$
 (5)

where,  $G(a_{PX})$  is given by the following equation.

$$G = a_{p,x} = \int H = a_{p,x} \cdot f = \sigma_0 \cdot d \cdot \sigma_0 \tag{6}$$

# Probabilistic Fatigue Life Evaluation of Proof Tested Sample

By above Eq(5), the correlation between residual equivalent crack size  $a_{Ps}$  and residual probability can be obtained. By using the equivalent crack size  $a_{Ps}$ , stress intensity factor was calculated by  $K = \sigma \sqrt{\pi a}$  equation for infinite plate.

For probabilistic fatigue life analysis, semi-elliptical surface crack of arbitrary aspect ratio was assumed as an initial crack. The initial crack size was determined by the following way: (1)Assume aspect ratio. (2)Determine crack size which maximum stress intensity factor at  $\sigma_{\max}$  is equal to the  $K_{\max}$  by using Newman-Raju equation[4], where

$$K_{max}$$
 is given by  $K_{max} = \sigma_{max} \sqrt{\pi a_{PX}}$ .

Subsequently fatigue life is predicted by using Paris power law and final failure condition was given by  $K_{max}\!\ge\!K_{1^{c}}$ .

## SPECIMENS AND EXPERIMENTAL PROCEDURE

### Specimen and Experimental Procedure

Sample is  $Si_3N_4$  sintered at  $1850^{\circ}$ C, in 1 atm  $N_2$  gas. This sample is not hot pressed, then it has considerably many flaws such as small pore. The sintered batch were cut into test pieces  $(0.8\text{mm} \times 10\text{mm} \times 100\text{mm})$ , because this size is sometime used as a plate spring. After cutting, surfaces of the test pieces were ground and polished before testing in accordance with the Japan Industrial Standard(JIS)[5], and final specimen's thickness was made to 0.8mm accurately.

The fracture strength was measured by a three-point bending test following the JIS method[5]. The span length and cross head speed were 30mm and 0.5mm/min, respectively. Fracture toughness was measured by the indentation method(load=49N) using Niihara's equation[6] for convenience. Proof test and fatigue test were carried out at room temperature in an air environment using mechanical fatigue testing machine. The both tests were carried out under deflection control mode. Fatigue test conditions were as follows: test frequency 10Hz, stress ratio R=0, and uniform moment zone is 10mm(wide)x20mm(length). Fatigue test was stopped when specimen didn't fracture up to 10 number of cycles. Proof test was carried out by stressing up to 880MPa as quick as possible to avoid a crack growth during proof test.

## RESULTS AND DISCUSSION

### Weibull Properties of Material Tested

Fracture stress  $\sigma_{\tau}$  is a material's constant and should be measured from the sample which has no obvious flaws on fracture surface. After bending test, fracture surfaces were investigated in detail using SEM. Twelve specimens were obtained by the test and Weibull properties of  $\sigma_{\tau}$  were listed in Table 1. Twenty  $K_{10}$  data were obtained by indentation method and their Weibull properties were also listed in Table 1.

### Fatigue Test Results

After cyclic fatigue test, the fracture surfaces were investigated in detail by using SEM. On fourteen specimens, fatigue crack was recognized to start from initial flaw, and the flaw shape could be defined clearly. From these specimens, initial flaws shape and aspect ratio were determined. From these data, it can be seen that most flaws were embedded one and their aspect ratio were almost about 1.0. However, their flaws shape are neither elliptic nor semi-elliptic and are very complex. It is very difficult or almost impossible to determine the K value for such flaws, exactly. In this paper, following simple method was adopted to determine the K value, for convenience. Semi-circular surface crack was assumed, which area is equal to real flaw area. Subsequently, initial stress intensity factor  $K_i$  of the semi-circular crack was calculated by using Newman-Raju Eq[4] and  $\sigma_{\max}$ .  $K_i$  versus number of cycle to failure  $N_f$  was plotted in Fig.2. The relation between  $K_i$  and  $N_f$  largely depends on material's constant C and m in Paris's power law  $\{da/dN = C(\triangle K)^m\}$ . Appropriate values were searched by try and error method, and finally.  $C = 1 \times 10^{-23}$  and m = 23 were determined as the best values as shown in Fig.2.

In Fig.3, fatigue test results on  $Si_3N_4$  spring model was shown. Solid symbols show the test results on no proof tested(virgin) specimen. Open symbols show the test results on proof tested specimen. Proof test has been made at the stress  $\sigma_P$  of 880MPa. This sample is not hot pressed and they have many flaws. Then their fatigue life showed very wide scatter. However, proof tested specimen show narrower scatter than virgin specimen. Lower band of  $\sigma_{max}$  -  $N_F$  curve for proof tested specimen is higher than that of virgin specimen by about 100MPa. From this figure, it can be concluded that proof test is useful for fatigue life. If proof stress  $\sigma_P$  is higher, the usefulness of the test will become more remarkable. However, if  $\sigma_P$  is settled at high level, few specimen will survives and cost will become high.

## Probabilistic Fatigue Life by Analysis

The correlation between probability  $G(a_{PX})$  and equivalent crack size  $a_{PX}$  was calculated by using Eq(6) and material's constants in Table 1, where  $G(a_{PX})$  is a residual probability

of equivalent crack larger than  $a_{PX}$ . By using the  $a_{PX}$  and Paris's power law, fatigue life can be calculated for the case of  $\sigma_P$  =880MPa as a function of  $\sigma_{max}$ , reliability and aspect ratio of initial surface crack. In Fig.4, calculated  $\sigma_{max}$  -  $N_f$  curves were shown as a function of reliability. This is a case of  $\sigma_P$ =880MPa and aspect ratio=1.0. These curves were made by the following ways: (a)Calculate  $a_{PX}$  as a function of probability as shown in Table 2. (b)Presume aspect ratio and determine the surface crack size which  $K_{max}$  is equal to that of equivalent crack  $a_{PX}$ . (c)Presume  $\sigma_{max}$  to some value. (d)Calculate  $N_f$  by using Paris's power law. From Fig.4, it can be seen that calculated probabilistic  $\sigma_{max}$  -  $N_f$  curves show very good agreement with experimental one.

#### **CONCLUSIONS**

Study has been made on the residual crack size distribution after proof test and usefulness of proof test on fatigue life of sintered  $Si_3N_4$ . The main results are as follows:

- (1) A new method is proposed to evaluate the residual crack size distribution after proof test by using process zone size failure criterion and two parameter Weibull distribution function of both  $|\sigma|_f$  and  $K_{1C}$ .
- (2) By fatigue test, it was well verified that proof test was very useful technology to guarantee fatigue life of ceramics members.
- (3) Probabilistic fatigue life of proof tested specimen was evaluated by using the above method proposed in this paper, calculated probabilistic fatigue life showed good agreement with experimental one.

#### REFERENCES

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TABLE 1- Two Parameter Weibull Coefficients of Fracture Stress  $-\sigma_{\rm f}$  and Plane Strain Fracture Toughness,  $K_{\rm fc}$ 

	Mean Value	Scale Parameter	Shape Parameter	
σr	1110 MPa	1150 MPa	17.2	
K <sub>IC</sub>	6.65 Mpa√ m	6.78 MPa√ m	18.4	

TABLE 2- Relationship between Residual Equivalent Crack Size  $a_{\scriptscriptstyle PN}$  and Reliability of Cyclic Fatigue Life

R(apy)	50 %	60 %	70 %	80 %	90 %
$a_{px} (\mu m)$	6.7104	7.1486	7.5935	8.0855	8.7183
$\frac{a_{\rm PX}}{a_{\rm PY}} \frac{\mu}{a_{\rm P}}$	1 0106	1 0766	1.1436	1.2177	1.3130

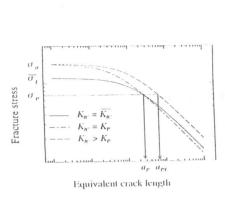


Figure 1 Correlation between fracture stress  $\sigma_{\rm C}$  and equivalent crack length, and also proof stress  $\sigma_{\rm P}$  and residual crack size  $a_{\rm P}$ 

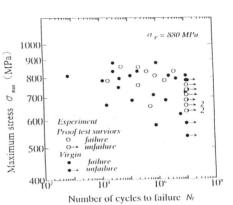


Figure 3 Relationship between cyclic maximum stress  $\sigma_{\rm max}$  versus number of cycle to failure  $N_{\rm f}$ 

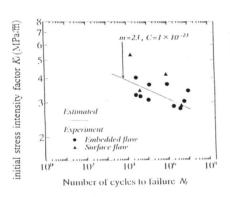


Figure 2 Correlation between initial stress intensity factor  $K_i$  and number of cycle to failure  $N_{\rm f}$ 

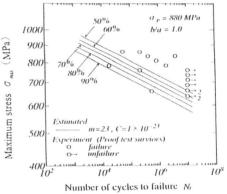


Figure 4 Comparison between calculated fatigue failure probability and experimental data